



1. Gericht geluid en kanaal-arrays
2. Massa-veer systemen

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LBP!SIGHT

Raadgevende ingenieurs

geluidbeheersing, bouwfysica, akoestiek, brandveiligheid

arbo, milieu en ruimtelijke ordening

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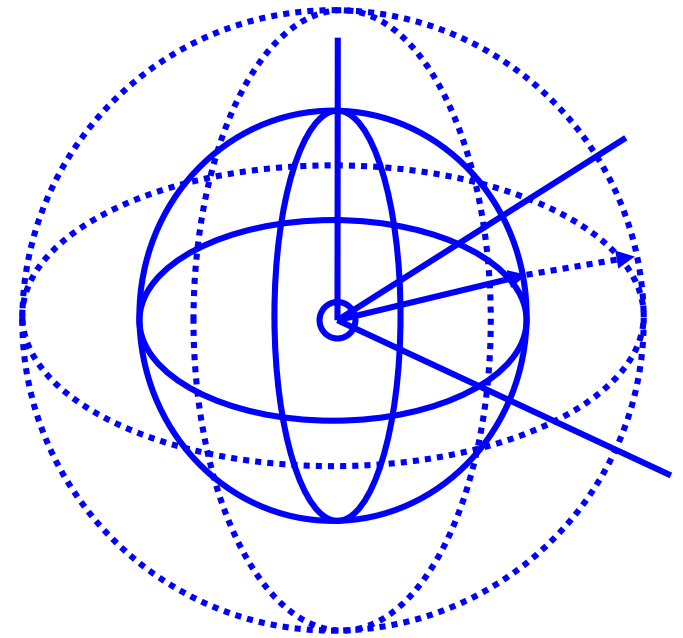
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1. RICHTEN VAN GELUID

- Eén bolbron: rondom geluid
- Twee bolbronnen in fase: interferentie

Bolvormige golven



Golfvergelijking:

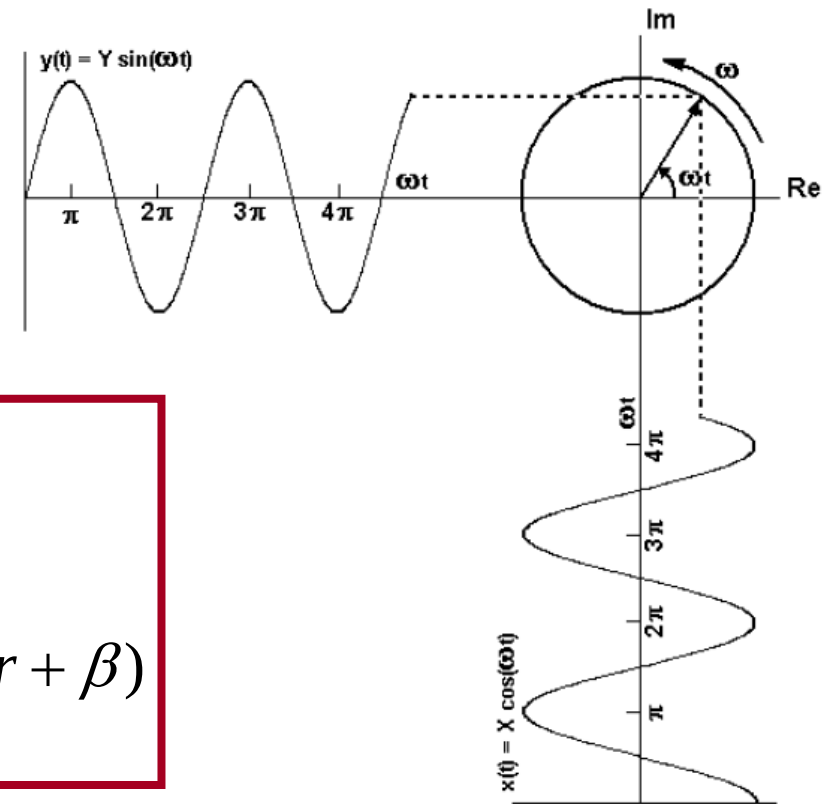
$$\frac{\partial^2(rp)}{\partial r^2} = \frac{1}{c^2} \cdot \frac{\partial^2(rp)}{\partial t^2}$$

Bolvormige golven

$$k \cdot r = \frac{2 \cdot \pi}{\lambda} \cdot r = \frac{2 \cdot \pi \cdot f \cdot r}{c} = \frac{\omega \cdot r}{c}$$

Harmonische oplossing

$$p = \frac{A}{r} \cdot \sin(\omega \cdot t - k \cdot r + \beta)$$



$$u = \frac{k \cdot A}{\omega \cdot \rho \cdot r} \cdot \sin(\omega \cdot t - k \cdot r + \beta) - \frac{A}{\omega \cdot \rho \cdot r^2} \cdot \cos(\omega \cdot t - k \cdot r + \beta)$$

Bolvormige golven

impedantie voor harmonische oplossing

$$\frac{p}{u} = \rho \cdot c \cdot \frac{j \cdot k \cdot r}{1 + j \cdot k \cdot r}$$

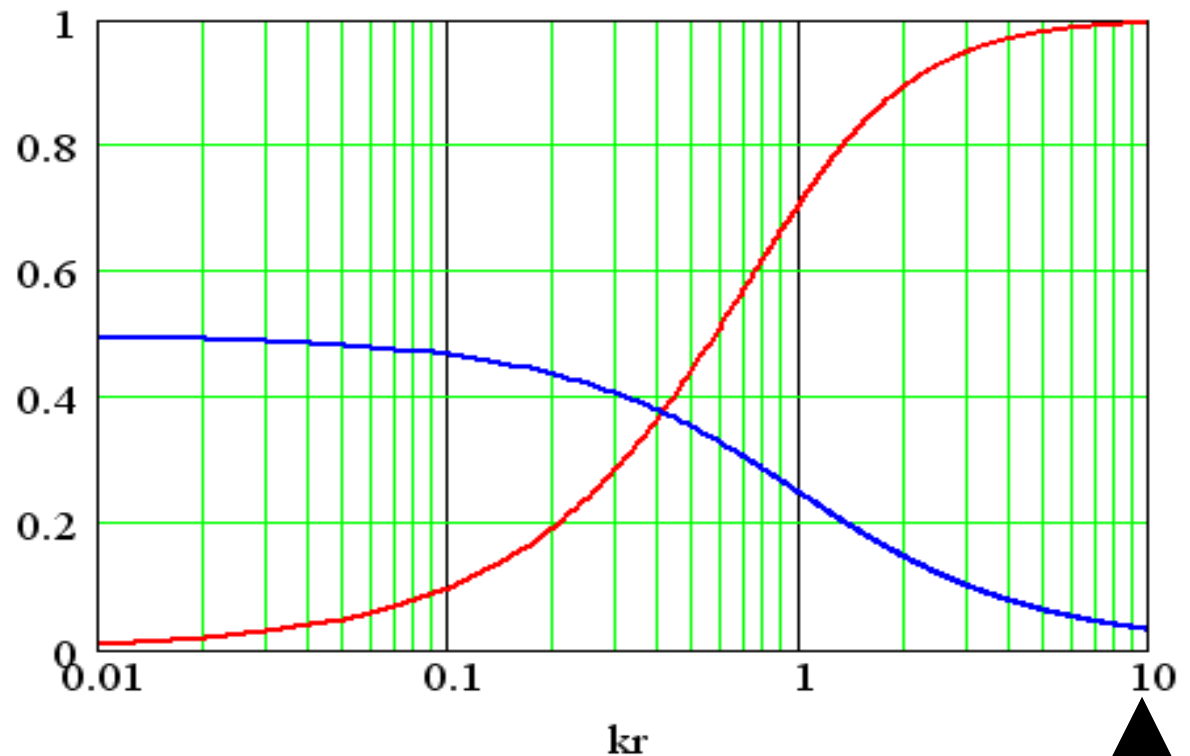
$$k \cdot r = \frac{2 \cdot \pi \cdot r}{\lambda}$$
$$r = \lambda \text{ dan: } k \cdot r = 2 \cdot \pi \approx 6,28$$

Bolvormige golven: impedantie

logaritmische schaal voor kr

amplitude $\left| \frac{j \cdot kr}{1 + j \cdot kr} \right|$

fase (rad/ π) $\frac{1}{\pi} \operatorname{atan} \left(\frac{1}{kr} \right)$



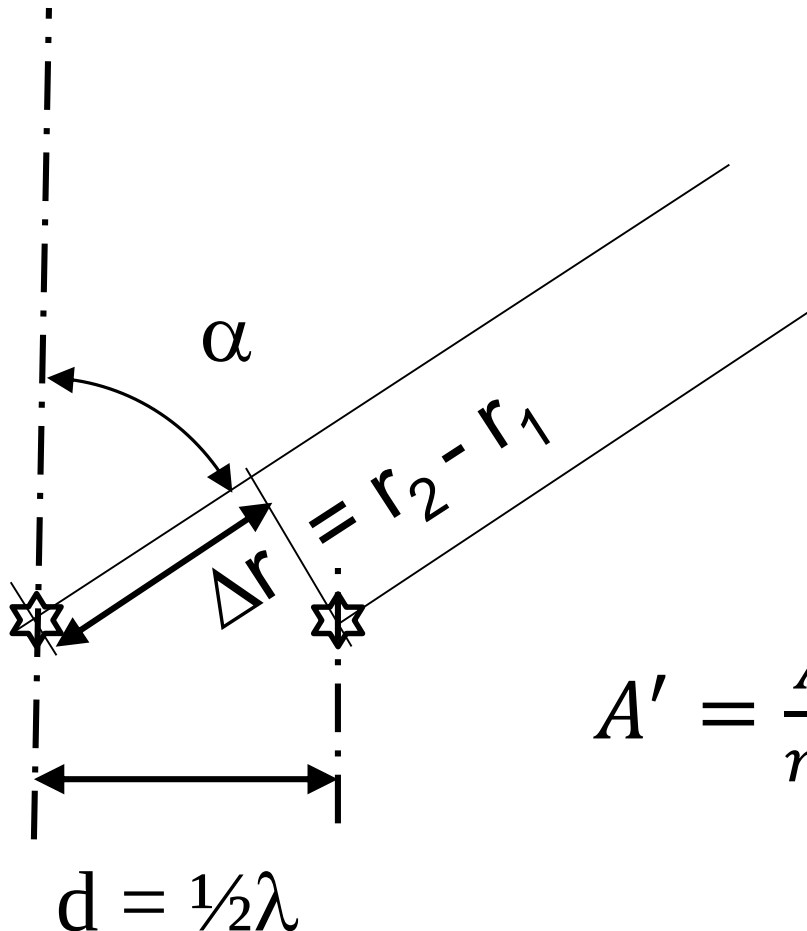
“Verre veld” geldt als: $r = 1,5 \cdot \lambda$ dan: $k \cdot r \approx 10$

Richtwerking 2 identieke bronnen op halve golflengte

Op grote afstand:

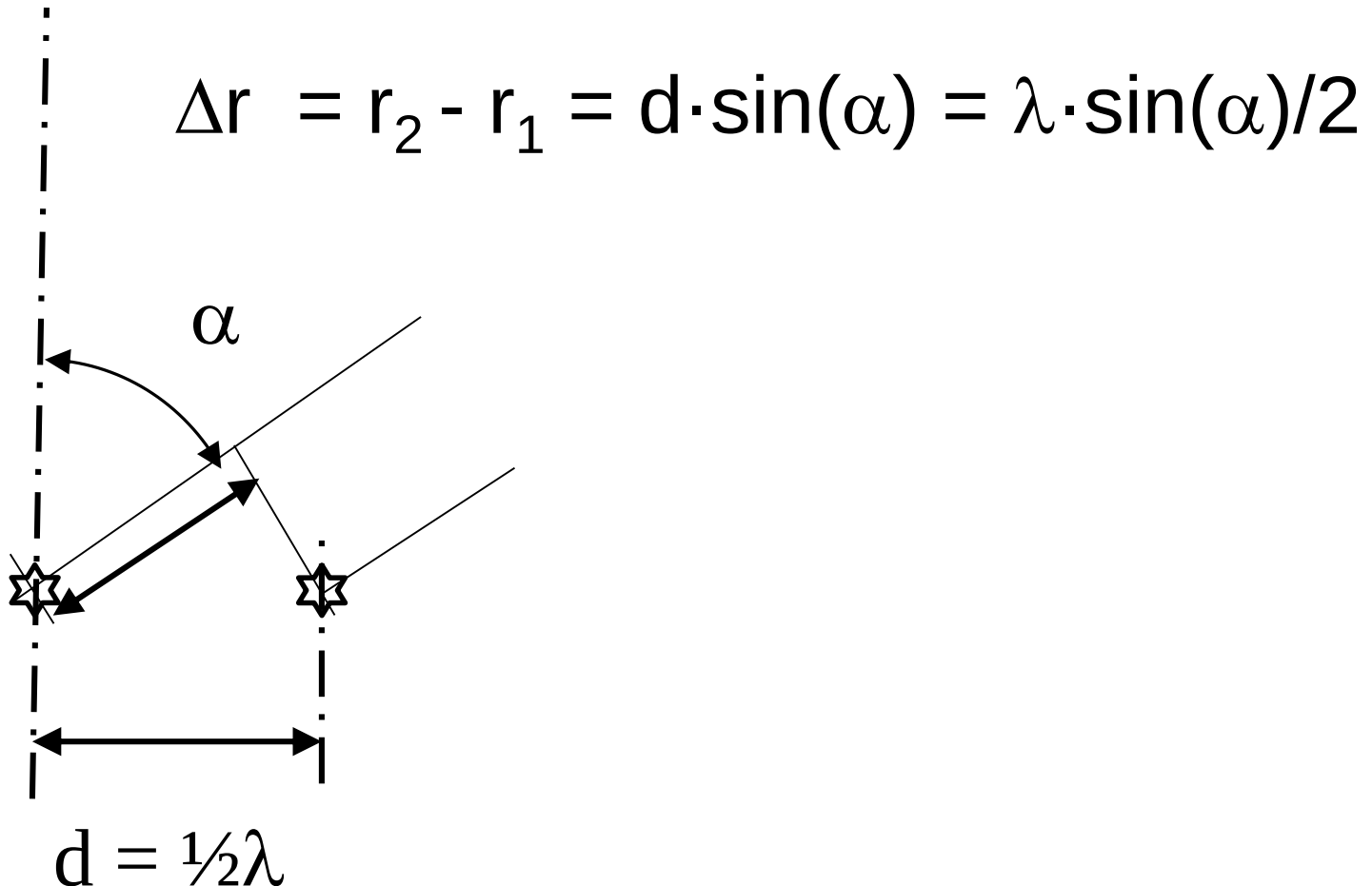
$$p_2 = A'' \cdot e^{j \cdot (\omega t - k r_2)}$$

$$p_1 = A' \cdot e^{j \cdot (\omega t - k r_1)}$$



$$A' = \frac{A}{r_1} ; A'' = \frac{A}{r_2} ; A'' \approx A'$$

Richtwerking 2 identieke bronnen op halve golflengte



Richtwerking 2 identieke bronnen op halve golflengte

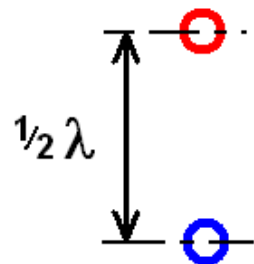
$$p = p_1 + p_2 = A \cdot (e^{j \cdot (\omega t - kr_1)} + e^{j \cdot (\omega t - kr_2)}) =$$
$$= p_1 \cdot (1 + e^{-j \cdot (\Delta r_2)}) \quad \dots\dots\dots$$

$$|p| = A \cdot \sqrt{2 + 2 \cdot \cos(\pi \cdot \sin(\alpha))}$$

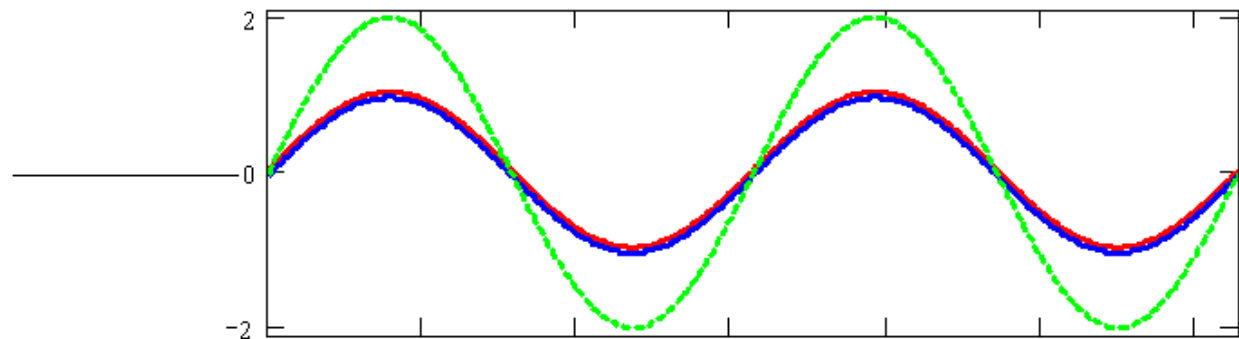
$$\alpha = 0 \quad : \quad |p| = 2A \quad \text{in fase optelling}$$

$$\alpha = \pi/2 : \quad |p| = 0 \quad \text{volledige uitdoving}$$

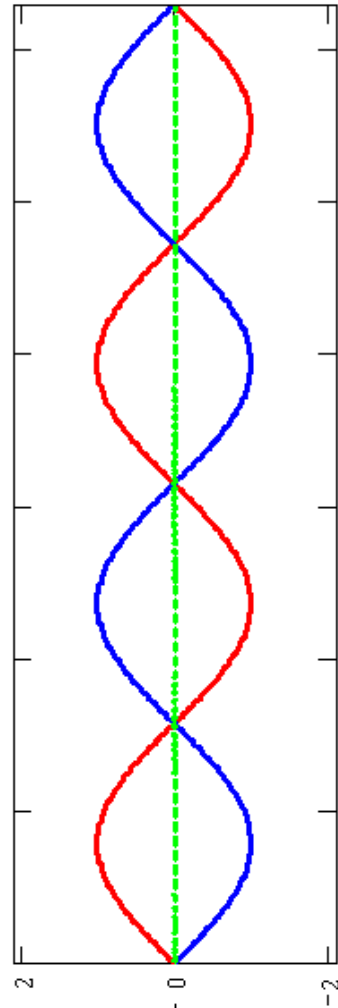
De werking:



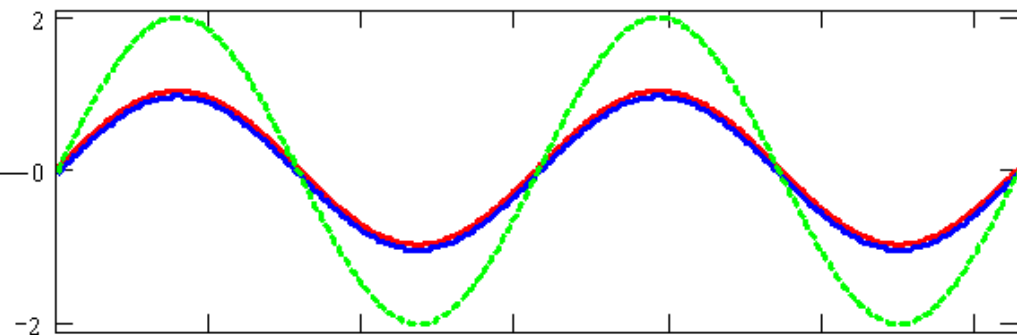
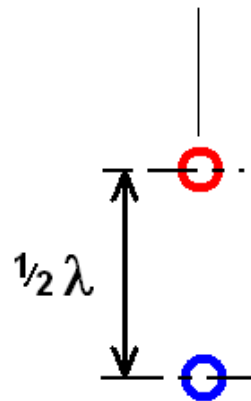
Red: from opening 1
Blue: from opening 2
Green: sum



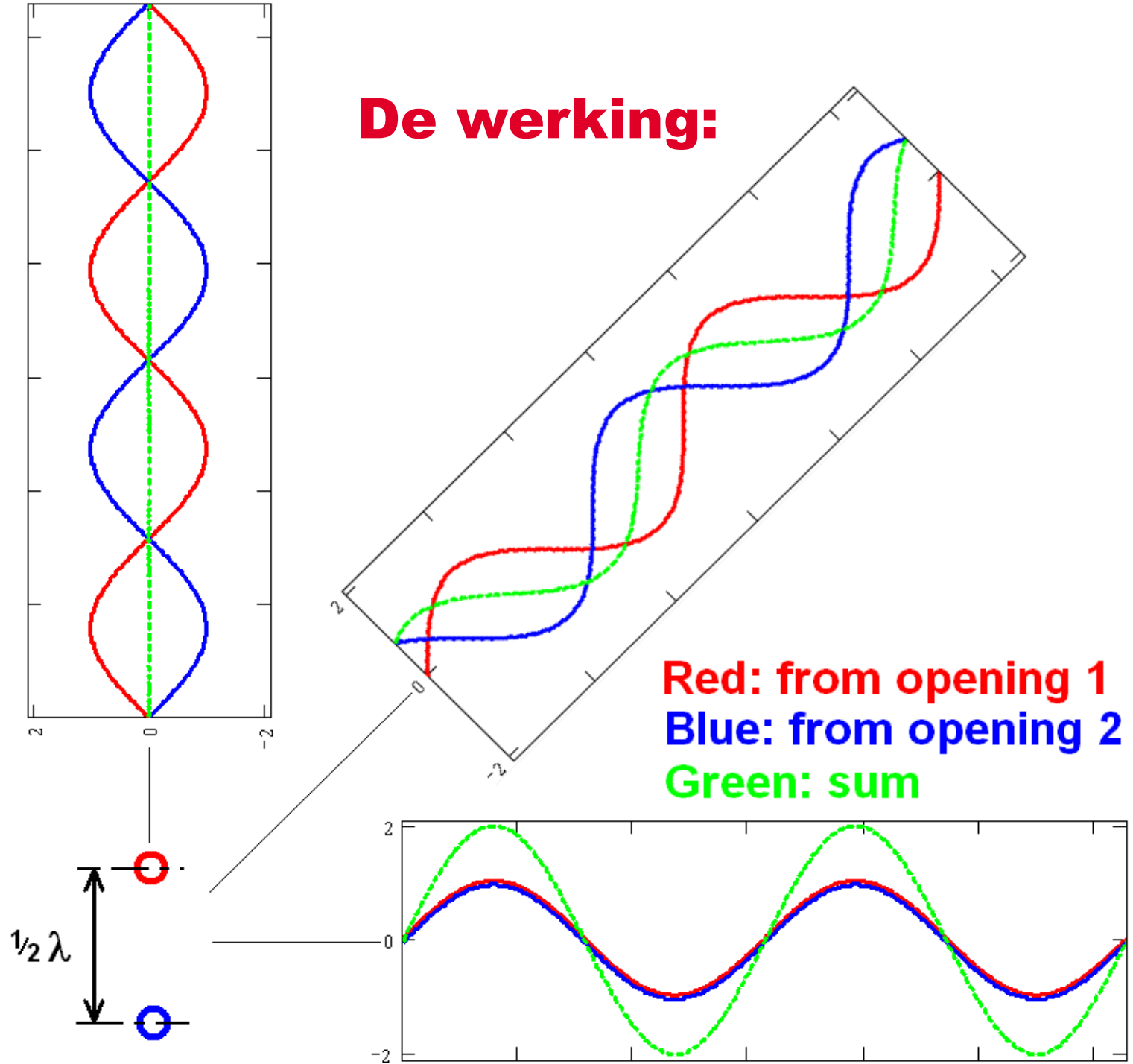
De werking:



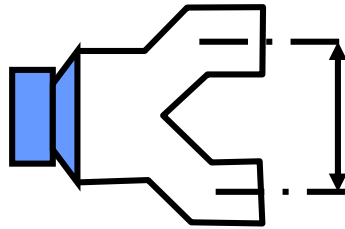
Red: from opening 1
Blue: from opening 2
Green: sum



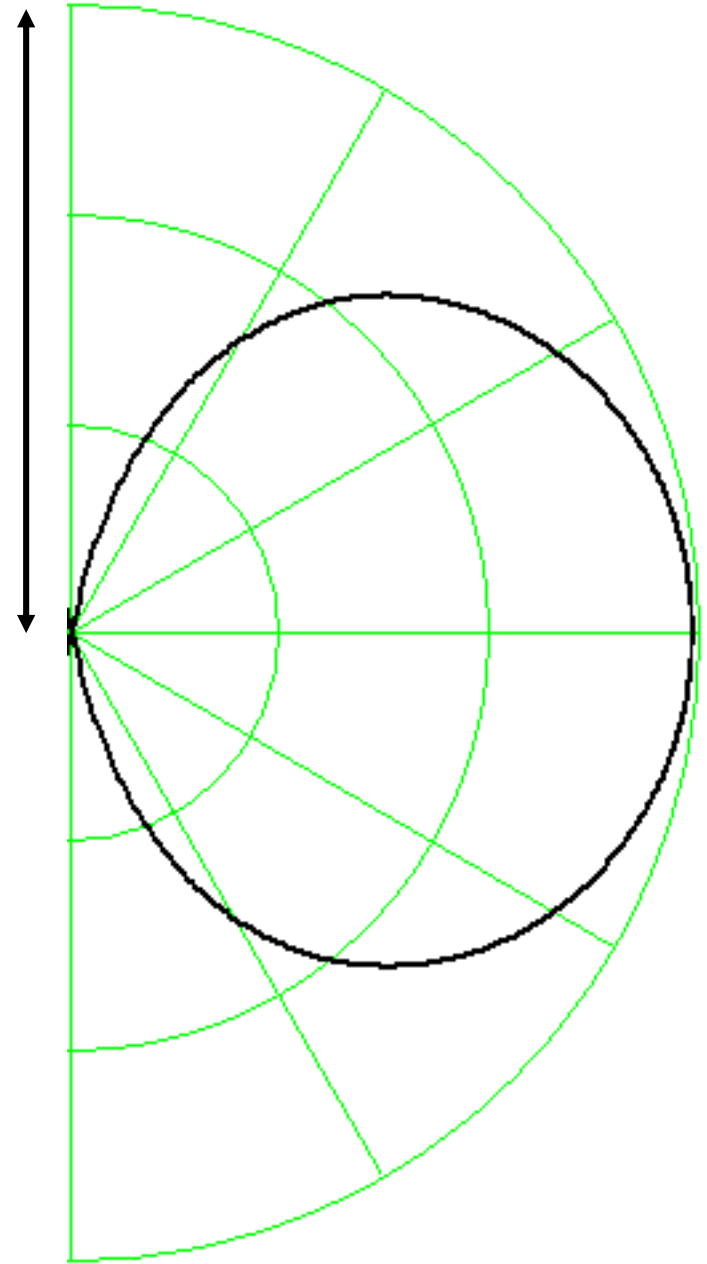
De werking:



Richtwerking bij 2 openingen



30 dB



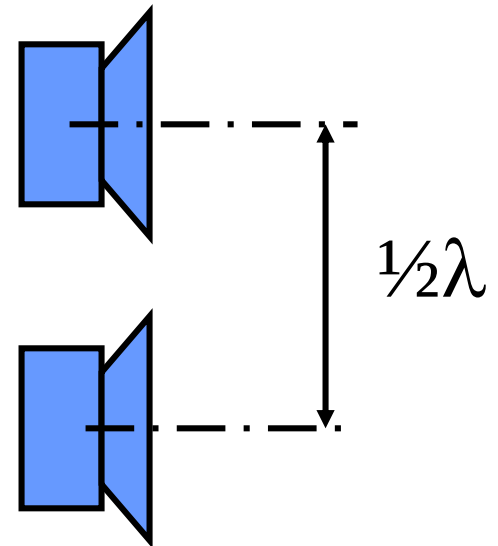
Richtwerking voor verminderen hinder

■ Hoe richten?

- luidspreker zelf?
- afschermende plaat?
- parabool?

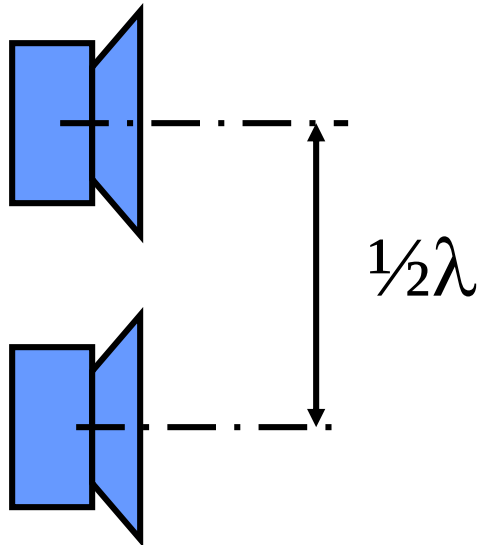
■ twee fase-gerelateerde bronnen nabij elkaar:

- hoe fase zekerstellen?
- regel-electronica, etc.

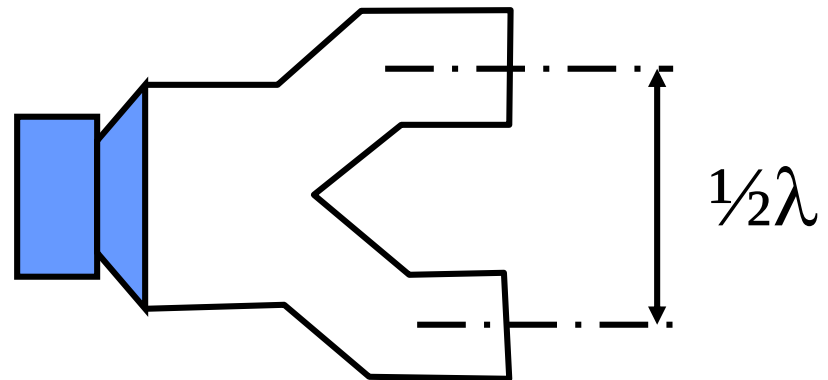


De $\frac{1}{2} \lambda$ richter

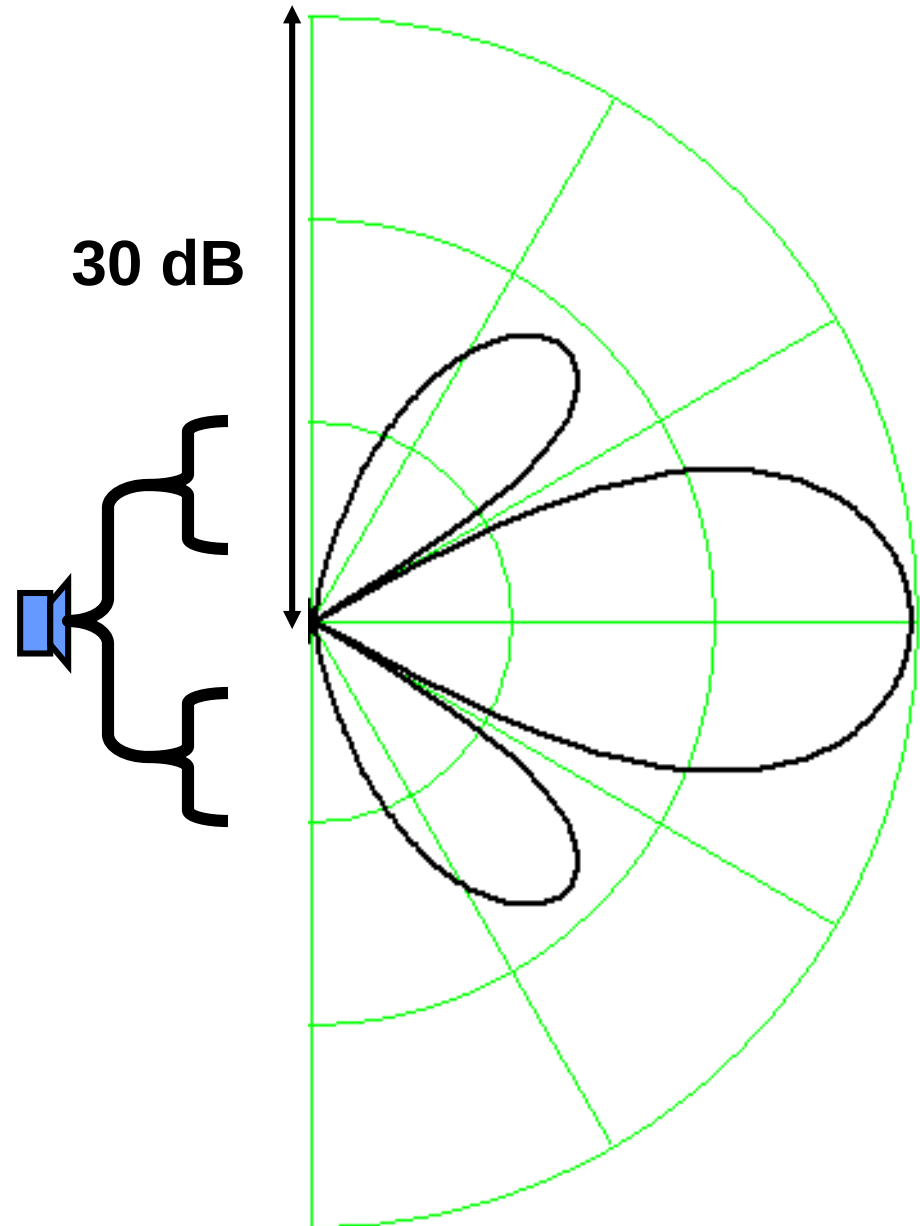
■ Bekend:



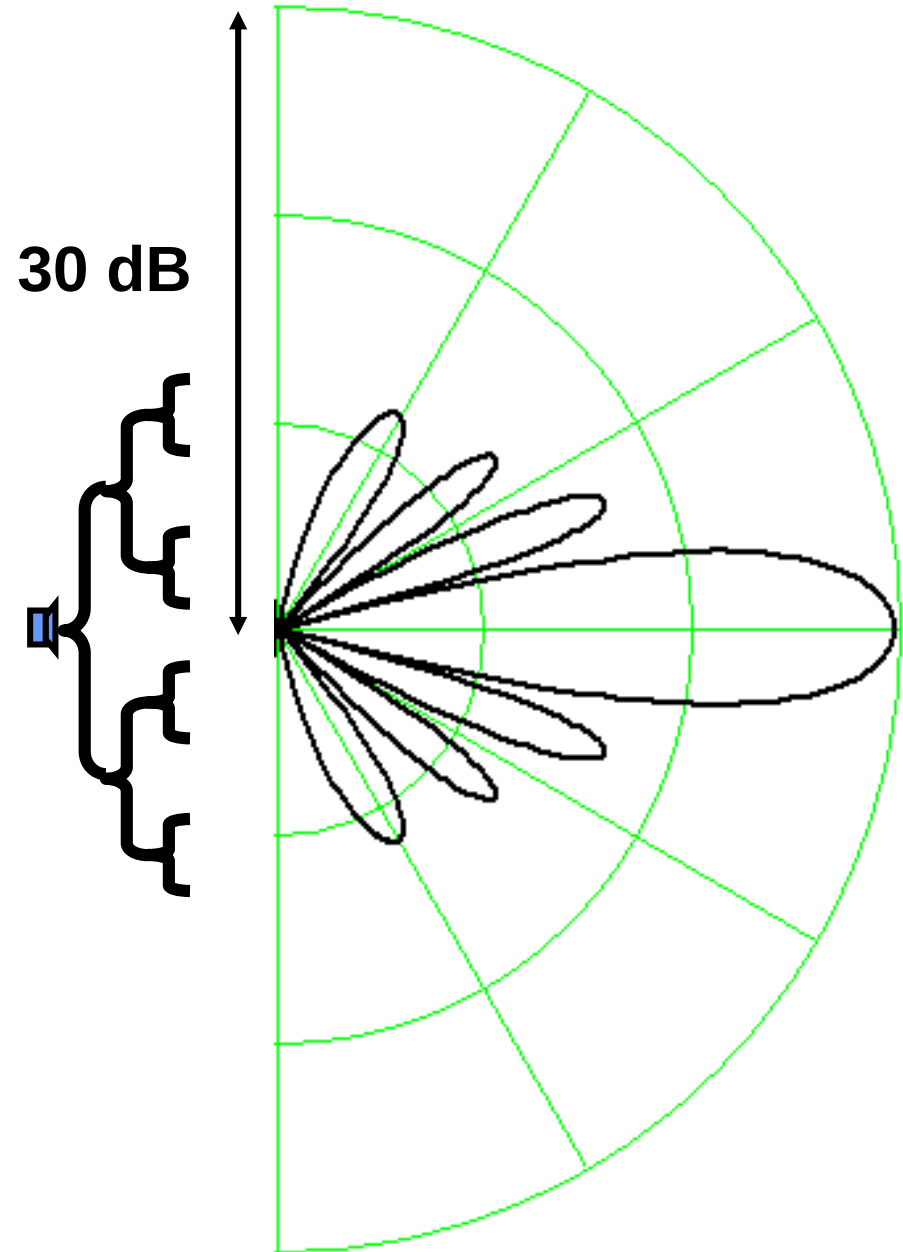
■ Kanaal-array:



Richtwerking bij 4 openingen



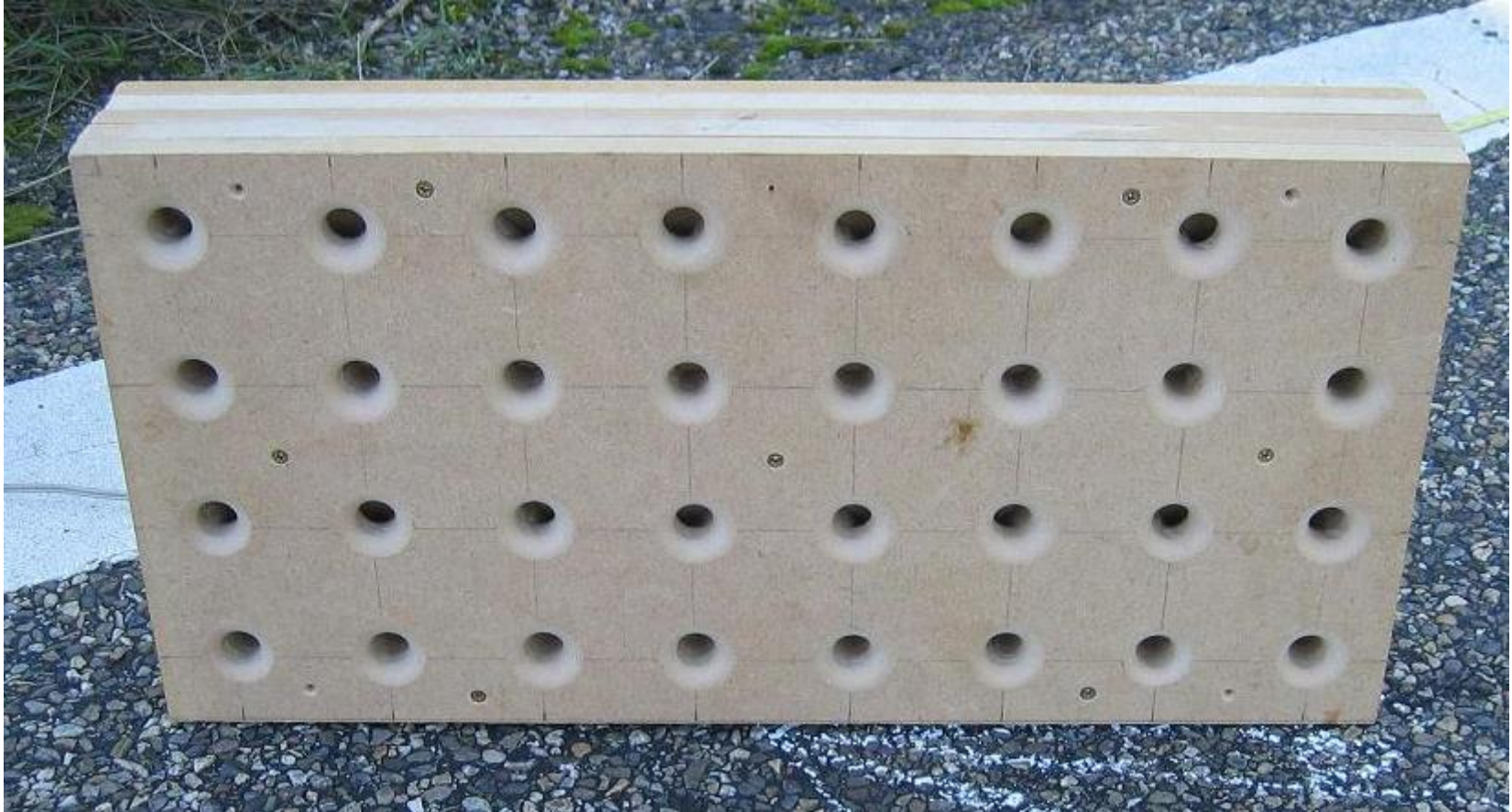
Richtwerking bij 8 openingen



Demonstratie

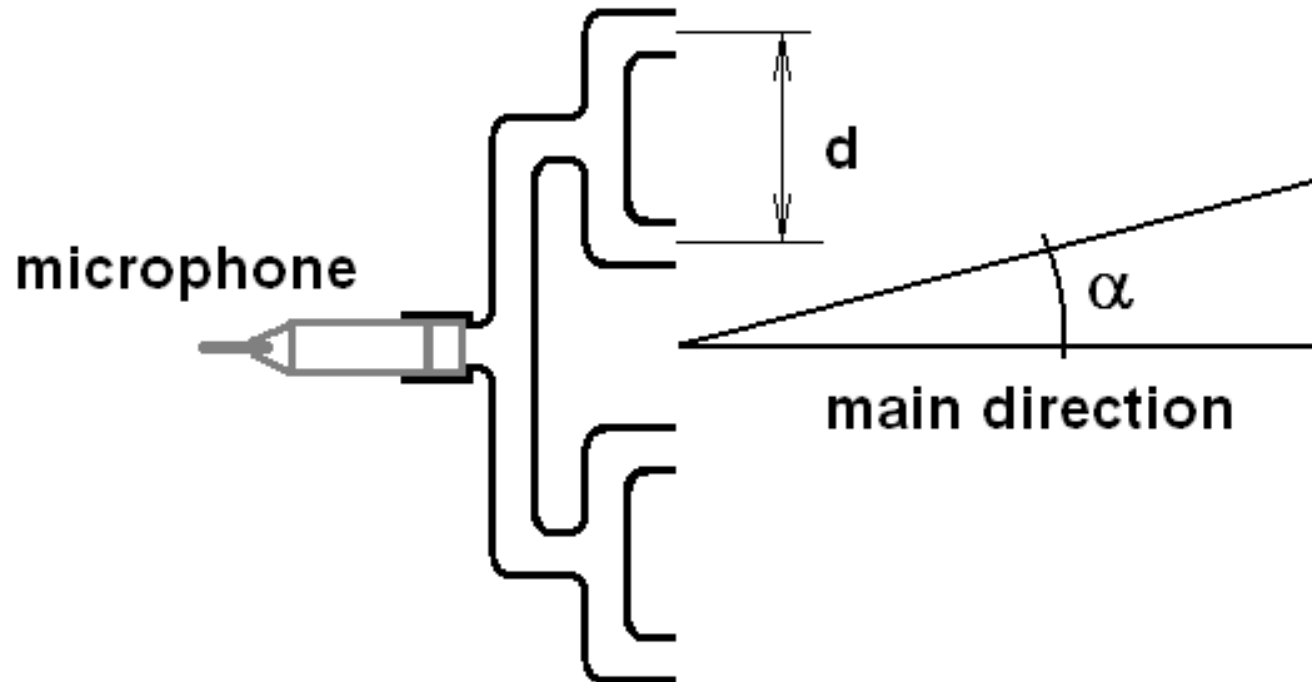


Demonstratie



Richtmicrofoon

gericht zenden = gericht ontvangen

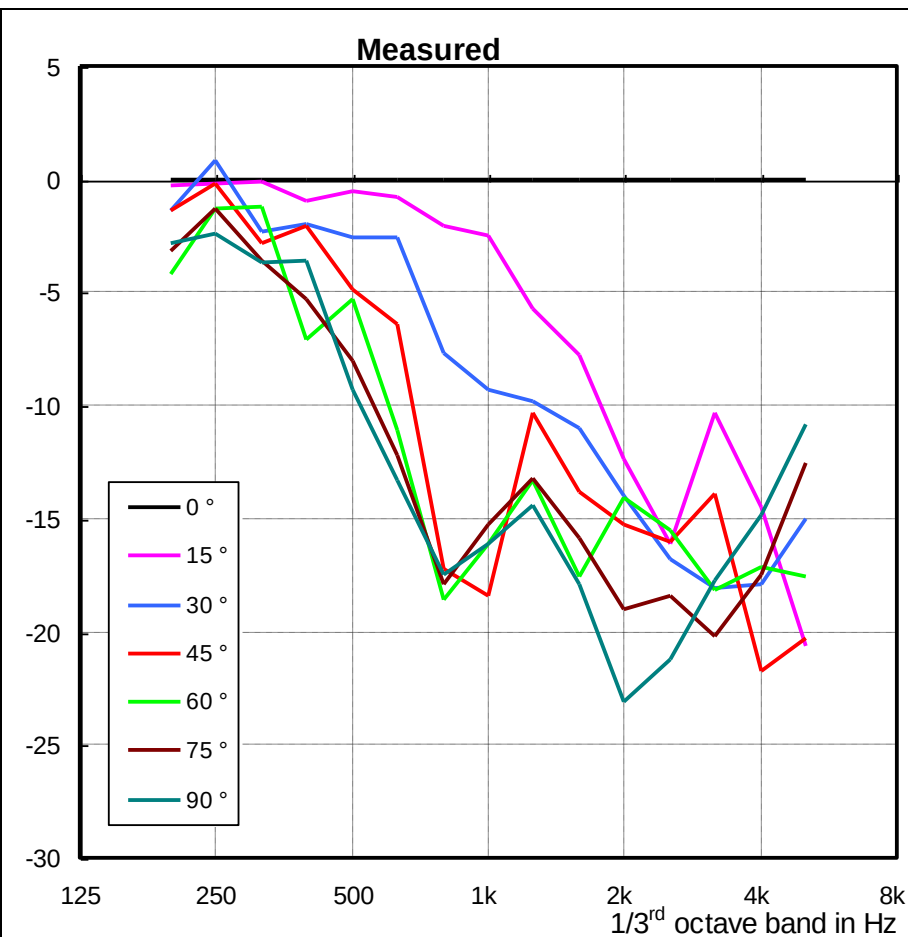
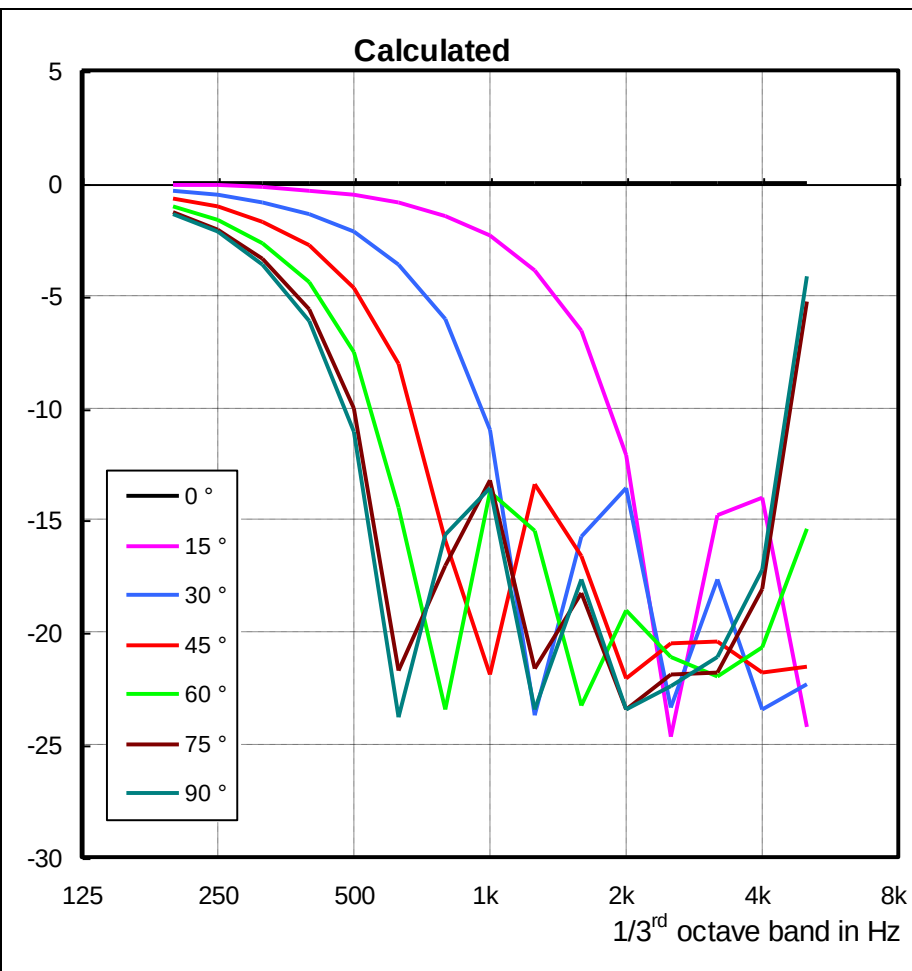


Prototype



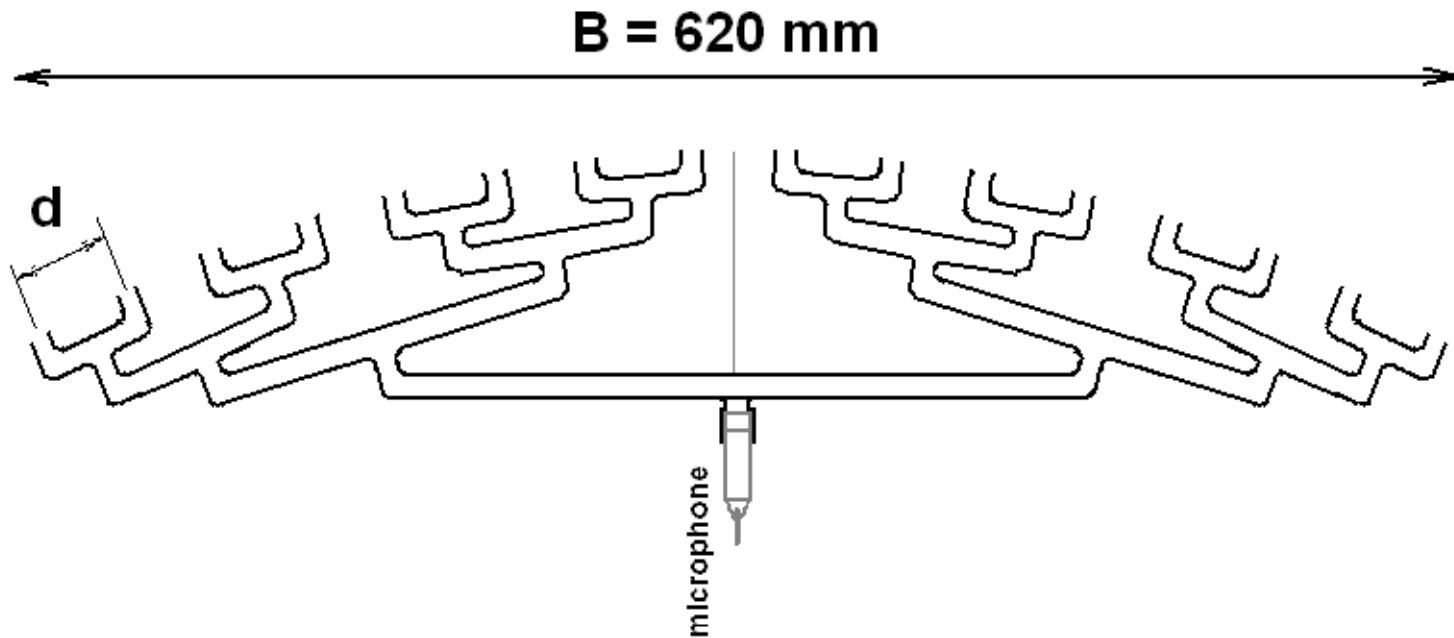
Theorie vs. meting: tertsbanden

Alle hoeken

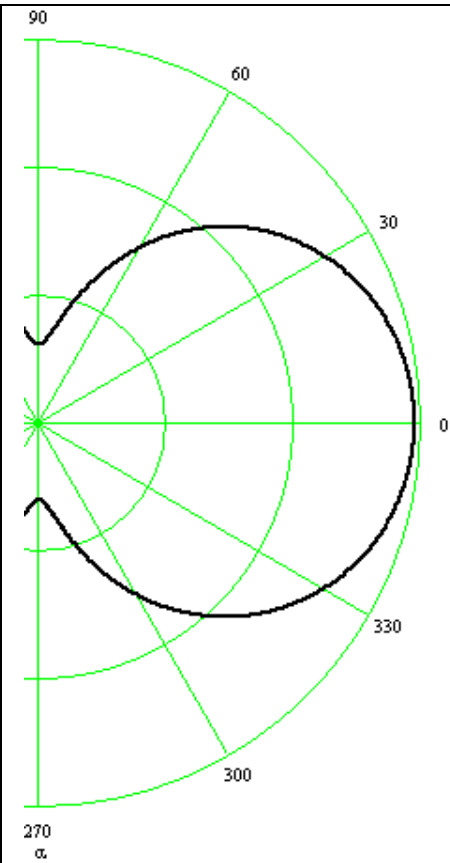


Verdere optimalisatie:

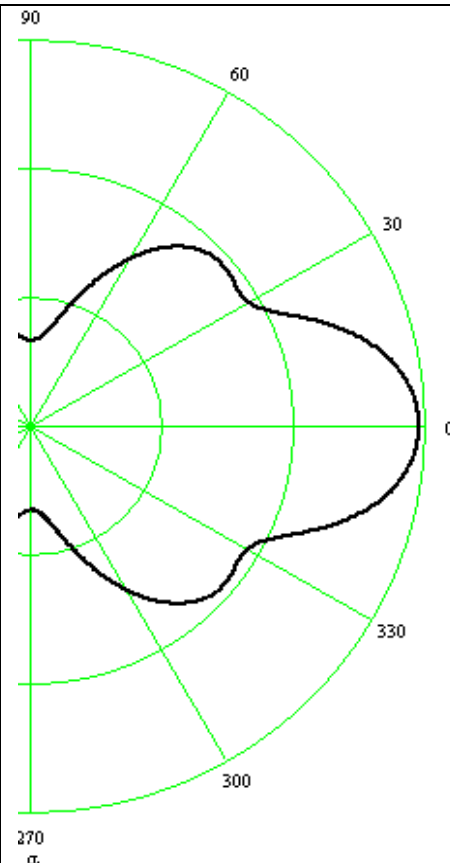
- 16 openingen
- Afstand tussen openingen: $d = 43 \text{ mm}$
- op cirkel met straal $0,573 \text{ m}$



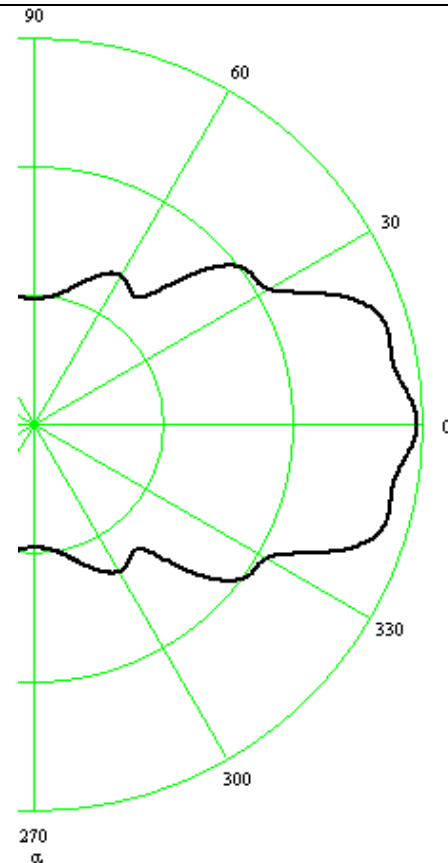
Theoretisch gedrag:



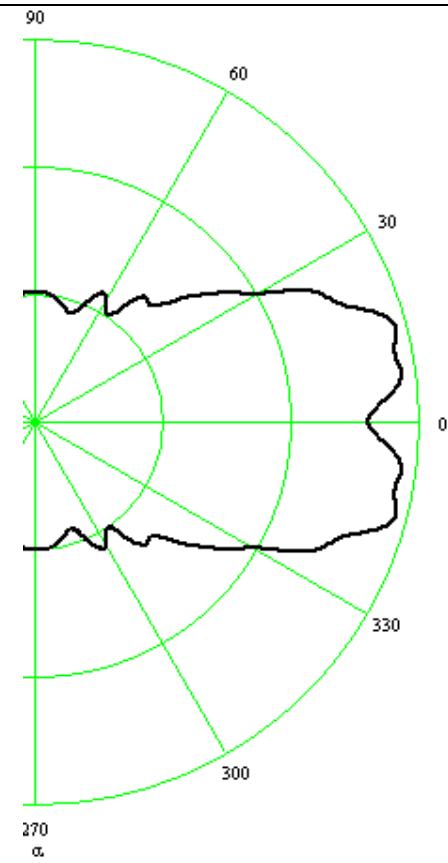
e. openings on curve
500 Hz 1/3rd octave



f. openings on curve
1000 Hz 1/3rd octave



g. openings on curve
2000 Hz 1/3rd octave

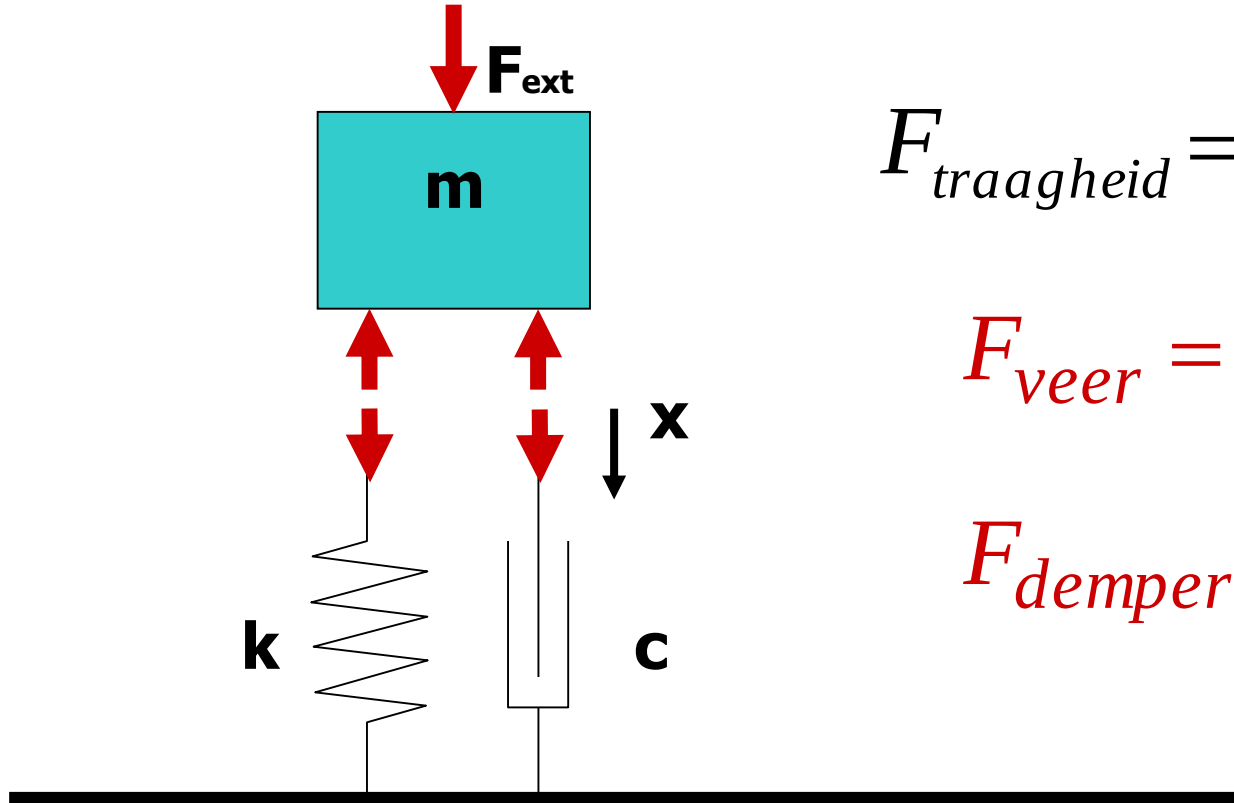


h. openings on curve
4000 Hz 1/3rd octave

2. MASSA VEER SYSTEEM

- Isoleren van trillingen van een machine naar vloer
- Isoleren van trillingen van vloer naar machine

2. Massa-veer systeem



$$F_{traagheid} = m \cdot \frac{d^2 x}{dt^2}$$

$$F_{veer} = k \cdot x$$

$$F_{demper} = c \cdot \frac{dx}{dt}$$

formule (krachten op massa M)

$$m \cdot \frac{d^2 x}{dt^2} + c \cdot \frac{dx}{dt} + k \cdot x = F_{ext}(t)$$

Uitwerking

stel: $x(t) = X \cdot e^{j\omega \cdot t}$ en $F_{ext}(t) = F \cdot e^{j(\omega \cdot t + \phi)}$

dan: $\frac{dx}{dt} = X \cdot j\omega \cdot e^{j\omega \cdot t} = j\omega \cdot x(t)$

en: $\frac{d^2x}{dt^2} = X \cdot j^2 \omega^2 \cdot e^{j\omega \cdot t} = -\omega^2 \cdot x(t)$

$$F_{ext}(t) = -\omega^2 \cdot m \cdot X \cdot e^{j\omega \cdot t} + j \cdot \omega \cdot c \cdot X \cdot e^{j\omega \cdot t} + k \cdot X \cdot e^{j\omega \cdot t}$$

$$= \left[k - \omega^2 \cdot m + j \cdot (\omega \cdot c) \right] \cdot X \cdot e^{j\omega \cdot t} = F \cdot e^{j(\omega \cdot t + \phi)}$$

$$F_{ext}(t) = -\omega^2 \cdot m \cdot X \cdot e^{j \cdot \omega \cdot t} + j \cdot \omega \cdot c \cdot X \cdot e^{j \cdot \omega \cdot t} + k \cdot X \cdot e^{j \cdot \omega \cdot t}$$

$$= [k - \omega^2 \cdot m + j \cdot (\omega \cdot c)] \cdot X \cdot e^{j \cdot \omega \cdot t} = F \cdot e^{j \cdot (\omega \cdot t + \phi)} = F \cdot e^{j \cdot \phi} \cdot e^{j \cdot \omega \cdot t}$$

delen door $e^{j \cdot \omega \cdot t}$ geeft:

$$[-\omega^2 \cdot m + \omega \cdot (j \cdot c) + k] \cdot X = F \cdot e^{j \cdot \phi}$$

vrije trilling, geen kracht, dus: $F = 0$,

dus dan oplossen: $-\omega^2 \cdot m + \omega \cdot (j \cdot c) + k = 0$,

ziet er uit als $\tilde{a} \cdot x^2 + \tilde{b} \cdot x + \tilde{c} = 0$ ($x = \omega$).

toepassen wortelformule $x_{1,2} = \frac{-\tilde{b} \pm \sqrt{\tilde{b}^2 - 4 \cdot \tilde{a} \cdot \tilde{c}}}{2 \cdot \tilde{a}};$

invullen van $\tilde{a} := m; \tilde{b} := -j \cdot c; \tilde{c} := -k$

Eigenfrequentie

$$\omega_{1,2} = \frac{j \cdot c \pm \sqrt{-c^2 + 4mk}}{2m}$$

Stel: geen demping, $c = 0$:

$$\omega_{1,2} = \frac{\pm \sqrt{4mk}}{2m} = \pm \sqrt{\frac{k}{m}} \quad \text{ofwel: } f_{1,2} = \pm 2\pi \sqrt{\frac{k}{m}}$$

$$f = 2\pi \sqrt{\frac{k}{m}}$$

Kritische demping

$$\omega_{1,2} = \frac{j \cdot c \pm \sqrt{-c^2 + 4mk}}{2m}$$

Omslag als $-c^2 + 4mk = 0$

$$c_{\text{crit}} = 2\sqrt{m \cdot k}$$

$c < c_{\text{crit}}$: harmonische beweging

$c \geq c_{\text{crit}}$: niet harmonisch gedempt

Overdracht: isolatie

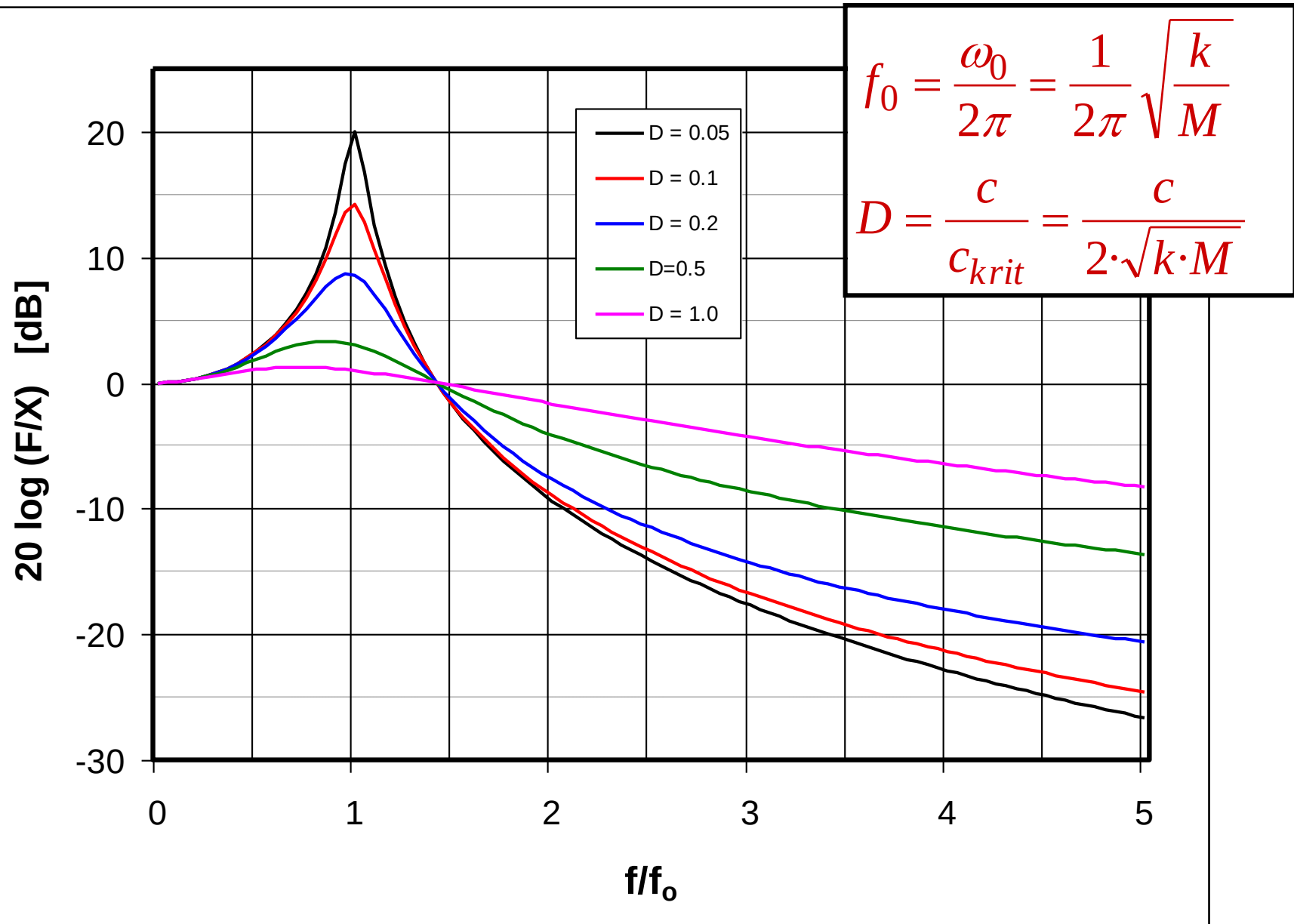
$$F \cdot e^{j\phi} = \left[k - \omega^2 \cdot M + j \cdot (\omega \cdot c) \right] \cdot X$$

daaruit volgt de overdracht:

$$\left| \frac{X}{F} \right| = \left| \frac{1}{k - \omega^2 \cdot M + j \cdot (\omega \cdot c)} \right| = \frac{1}{\sqrt{(k - \omega^2 \cdot M)^2 + (\omega \cdot c)^2}}$$

$$\phi = \arctan \left(\frac{\omega \cdot c}{k - \omega^2 \cdot M} \right)$$

Grafische weergave



Casus 1: Gasmotor



Casus 1: Maatregel

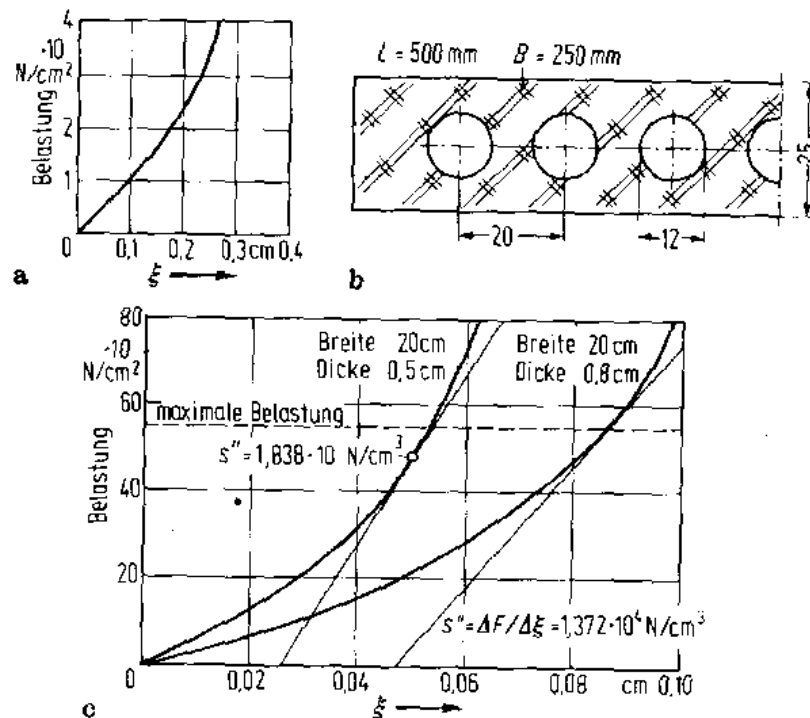


Bild 22.5. Federkennlinien von flächigen Lagerungen.

a, b gelochte Gummipatte; c ungelochte Gummipatte

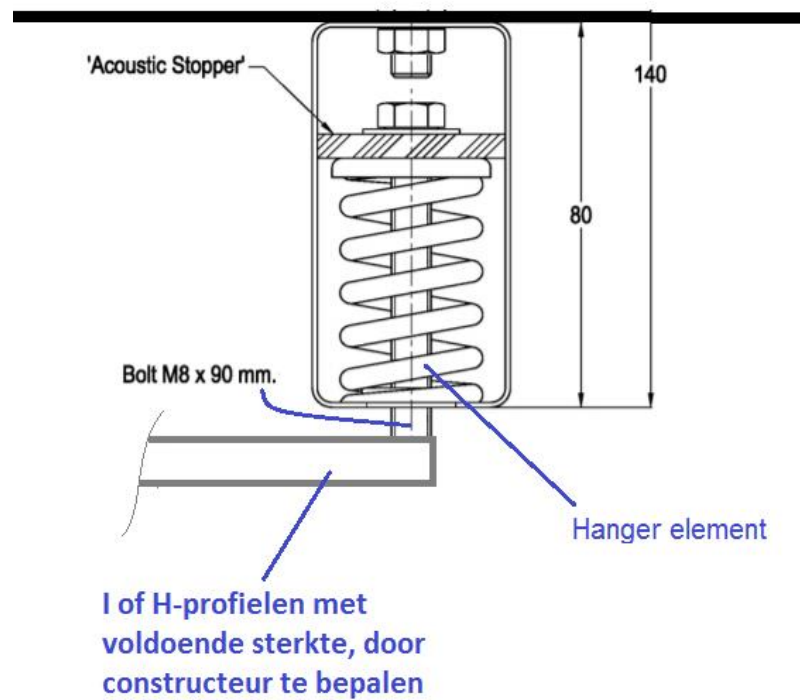
■ Taschenbuch der Technischen Akustik - Heckl & Müller

■ 2. Auflage

Casus 2: Ventilatoren parkeergarage



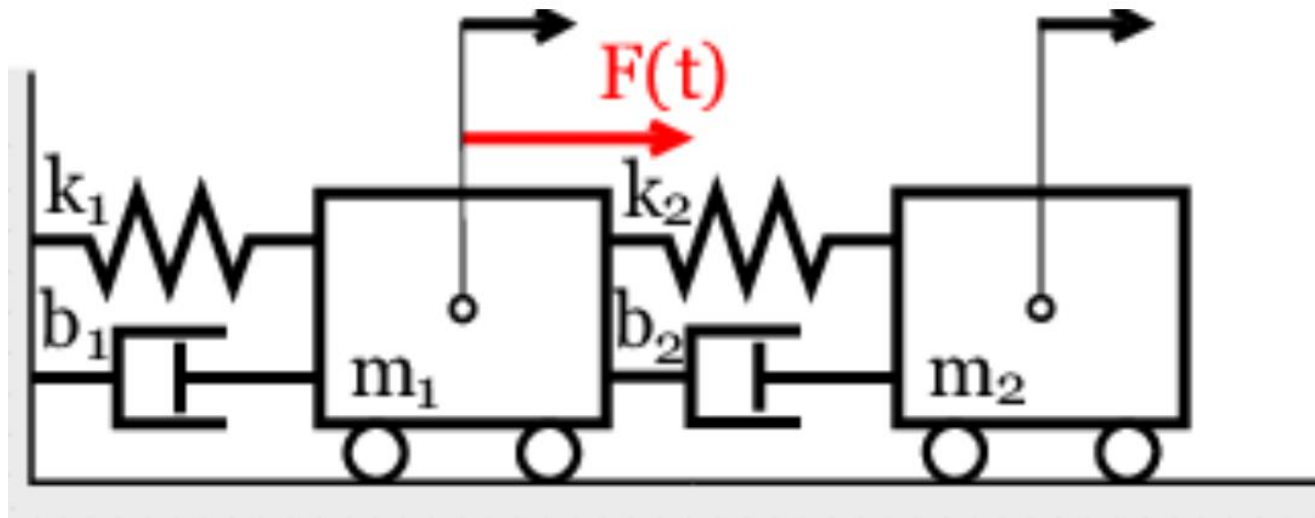
Casus 2: Ventilatoren parkeergarage



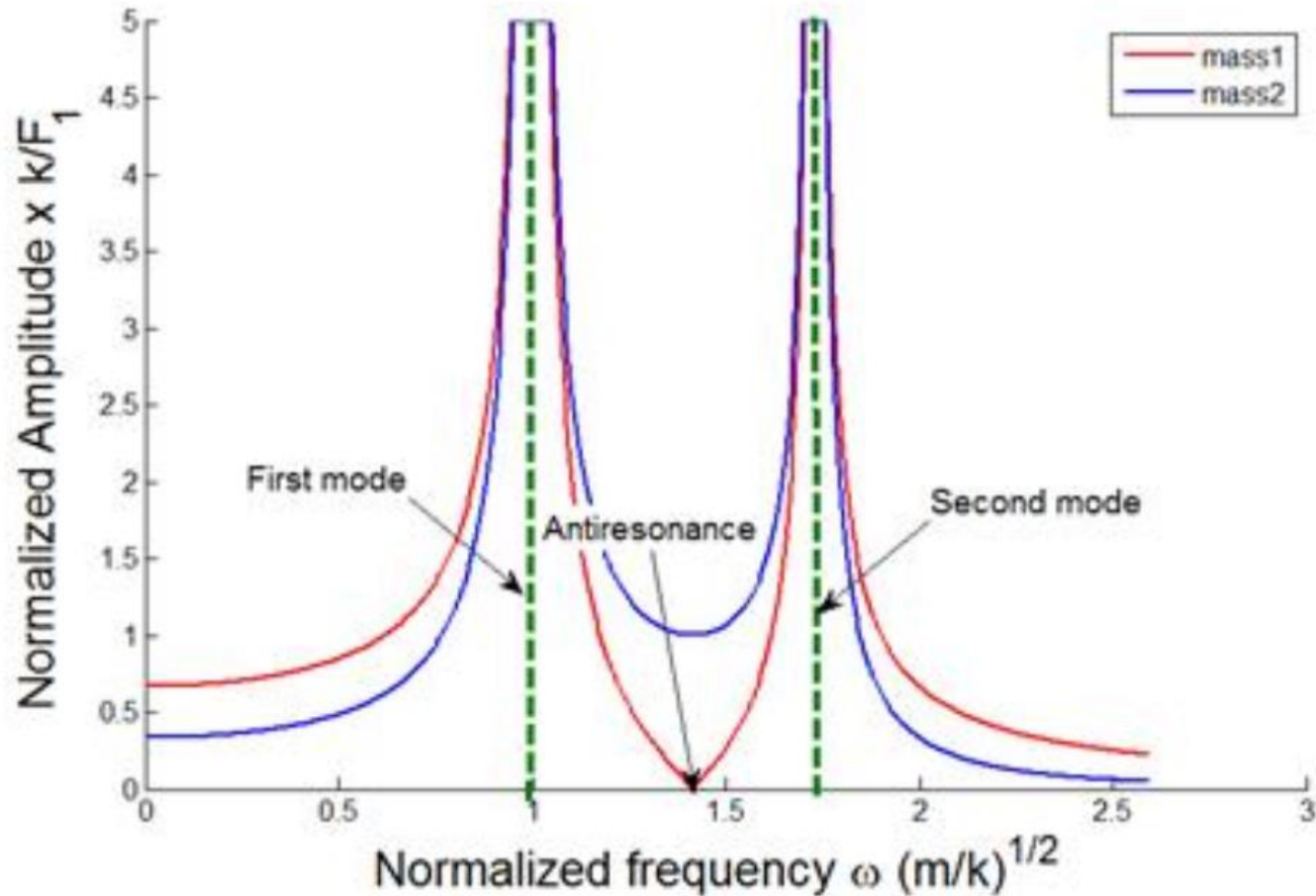
Casus 2: Ventilatoren parkeergarage



2 graden van vrijheid

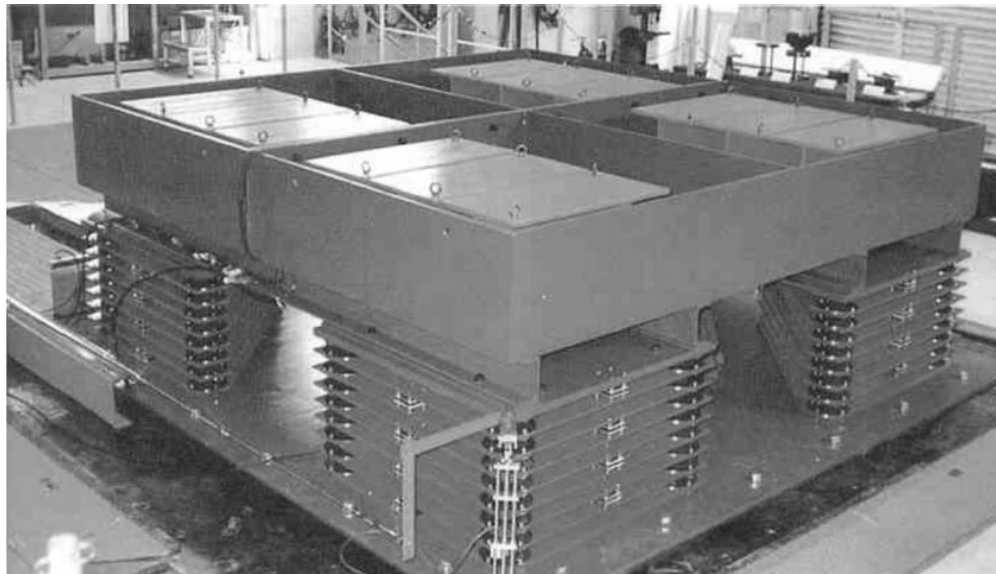


2 graden van vrijheid



2 graden van vrijheid

Toepassing: Tuned mass damper



Bron: J.Connor, Huis ten Bosch tower, Nagasaki



Dank voor uw aandacht

Vragen, bespreking, discussie

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